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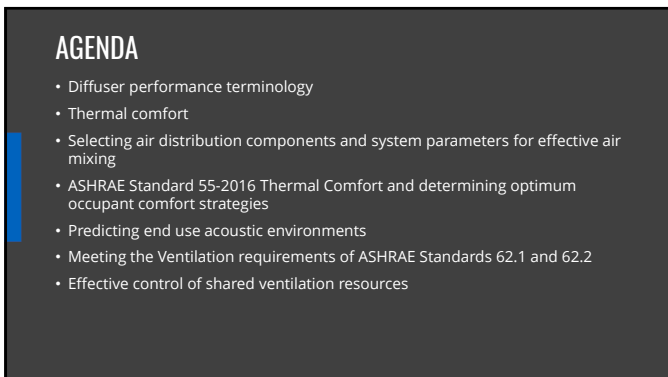
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## RESIDENTIAL, MULTI-FAMILY (Low, Medium and High Rise), AND COMMERCIAL

- Air distribution follows basic rules, whether an office or a residence
- The air distribution system in most single family dwellings, even expensive ones, is seldom highly engineered
- With multi-family dwellings, there are opportunities to distribute ventilation or heating/cooling flow within or between residences.
- Ventilation codes are requiring variable quantities of ventilation air as occupants run kitchen hoods and driers
- This implies that a shared ventilation supply needs to be dynamically controlled to be effective, which starts to closely resemble a commercial VAV system

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## TERMINOLOGY

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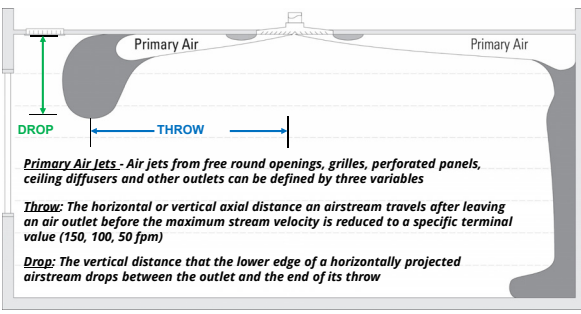
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## UNDERSTANDING THE TERMINOLOGY



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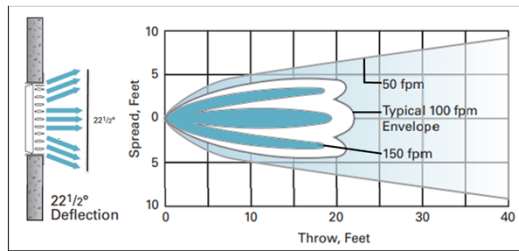
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## UNDERSTANDING THE TERMINOLOGY



**Spread** - The divergence of the airstream in a horizontal or vertical plane after it leaves the outlet

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## UNDERSTANDING THE TERMINOLOGY

Understanding **primary air jet** variables enables:

- Accurate prediction of room air flow
- Improvement of thermal comfort
- Proper selection of grilles, registers, and diffusers
- Making the best economic decision understanding the tradeoffs between cost and performance

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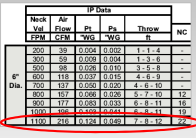
## UNDERSTANDING THE TERMINOLOGY

The Basis of Catalog Performance Data

- **Throw** - The horizontal or vertical axial distance an airstream travels after leaving an air outlet, usually assumes a surface adjacent to the air outlet
- **Pressure** - Can be total pressure or static pressure
- **Sound** - Can be either NC or Octave Band data

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## THROW



IP Data							
Neck	Vel	Flow	P <sub>s</sub>	P <sub>v</sub>	Throw	NC	
Size	ft/min	CFM	in. WG	in. WG	ft		
150	39	0.004	0.002	1.1-4			
100	58	0.009	0.004	1.1-6			
50	85	0.026	0.012	2.1-8			
150	113	0.037	0.013	4.1-9			
100	137	0.050	0.023	4.1-10			
50	167	0.066	0.028	7.1-10			
150	177	0.083	0.033	8.1-11			
100	204	0.099	0.040	8.1-11			
50	245	0.124	0.053	7.1-10			

150 fpm    100 fpm    50 fpm

7 - 8 - 12

- Throws are cataloged for 150, 100 and 50 fpm terminal velocities
- Throws should be selected so that jets do not collide, but have sufficient projection for the area to be served
- The air supply primary jet should never be directed at an occupant unless the outlet is highly adjustable

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## PRESSURE

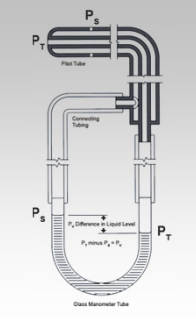



Diagram illustrating pressure measurement using a pitot-static probe and a U-tube manometer. The probe has static ports (P<sub>s</sub>) and a total pressure port (P<sub>T</sub>). The manometer shows the pressure difference between P<sub>s</sub> and P<sub>T</sub>.

- Pressure** – Air outlet pressure data is required to properly size the air delivery system within a building
- Static Pressure** – The outward force of air within a duct, measured in inches of water column
- Velocity Pressure** – The forward moving force of air within a duct, measured in inches of water column
- Total Pressure** – The sum of the velocity and static pressures, expressed in inches of water column and can be obtained by use of a pitot tube
- $P_T = P_V + P_S$

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## SOUND



Sound levels reported for diffusers are conducted in accordance with ASHRAE Standard 70

- Catalog sound data assumes several diameters of straight duct
- Room absorption is assumed to be 10 dB in all bands
- In practice however, room sound levels are probably 5 NC higher than reported

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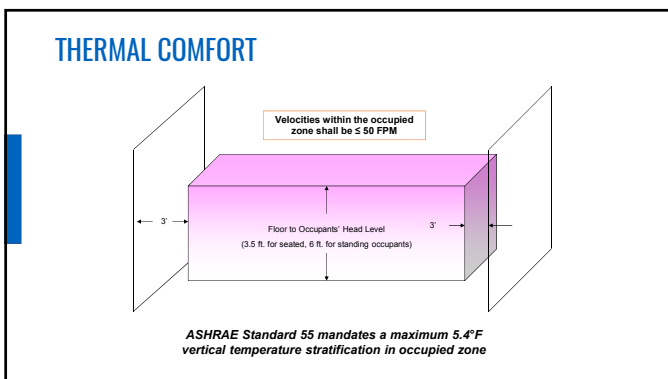
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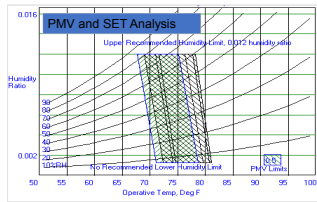
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## ADPI, LEED, AND THERMAL COMFORT



- 73°F (22.7°C) setpoint satisfies most Standard 55 input conditions
- Radiant effects are negligible (typically)

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## Poll #2

Certificate Password:

**HotAir19**

Note: This password is case sensitive



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## PROPER OVERHEAD HEATING DESIGN

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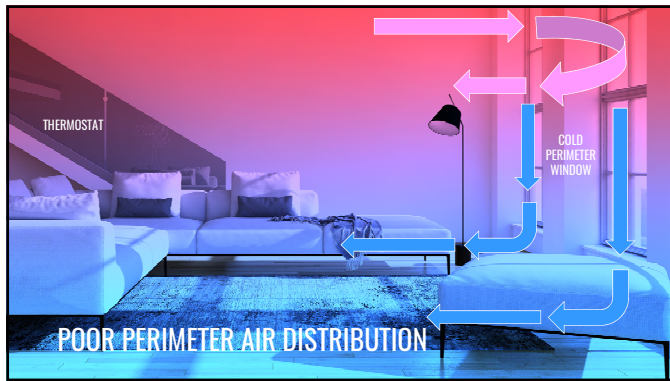
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### PERIMETER CONSIDERATIONS

- Maximum temperature difference between supply air and room temperature for effective mixing when heating, per ASHRAE handbook = 15°F (90°F discharge), continuous operation
- 150 FPM must reach 4.5 feet from the floor or less
- ASHRAE 62.1 requires that ventilation be increased by 25% when heating, if the above rules are not followed
- ASHRAE Handbook says that one should use linear diffusers, with throw toward and away from glass, to get acceptable performance in both heating and cooling
- Put a return slot above the window to carry away solar heat gain

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## NON-TYPICAL THROW ANALYSIS

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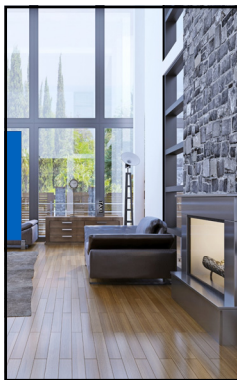
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### SPECIAL APPLICATIONS

#### High Bay Application - Ceilings Over 12' High

- Heating is a challenge due to buoyancy
  - Take advantage of vertical stratification where possible
  - Required Heating airflow rate may exceed cooling airflow rate
  - Keep heating supply air temperature to room temperature  $\Delta T$  to a minimum
- If supplying air distribution from the ceiling, consider using round diffusers, drum louvers, or diffusers with some vertical projection
- One cannot use ADPI to predict heating performance
- Consider Displacement Ventilation

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### DIFFUSER SELECTION & BUOYANCY

- ADPI isn't always the best way to analyze, select and place diffusers, especially with heating and high bay applications
- One can estimate throw as a function of  $\Delta T$  and buoyancy
- Simple rule: Distance to 75 ft/min is affected by 1% / degree(F)  $\Delta T$

#### Example:

1. 20° $\Delta T$  Cooling, Vertical Down = +20% projection
2. 20° $\Delta T$  Heating, Vertical Down = -20% projection
3. 20° $\Delta T$  Heating, Along Ceiling = +20% projection

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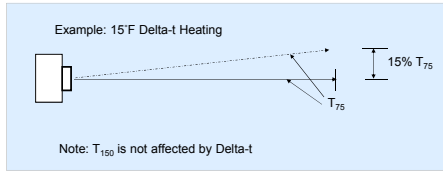
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## SIDE WALL REGISTER SELECTION AND BUOYANCY

### Horizontal Free Jet:

- Vertical change @ 75 ft/min is affected by 1% of 75 fpm throw/ $F^0 \Delta T$



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## ENTRAINED VS. FREE JETS

- Most catalog throw data assumes jet is along a surface
- Exceptions include drum louvers, duct mounted grilles and vertical linear diffusers
- A free jet will be shorter than an entrained jet because it has more surface area to induce surrounding air, which shortens throw



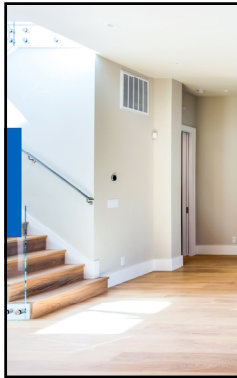
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## CONTINUOUS DUCT APPLICATIONS

- Use multiple drum louvers, duct mounted grilles, or continuous linear applications (longer than 10')
- Size duct as large as possible (duct inlet velocity < 1000 fpm)
- If inlet velocities are < 1000 fpm, maintain constant duct size through entire length of run and balancing will be minimal

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## RETURNS

- Typically located in/near the ceiling
- Returns have an almost immeasurable effect on room air flows < 1.5 cfm/sf
- Suspended ceilings typically leak 1 cfm/sf at 0.1" differential pressure
- Spaces with high airflow rates can benefit from low returns

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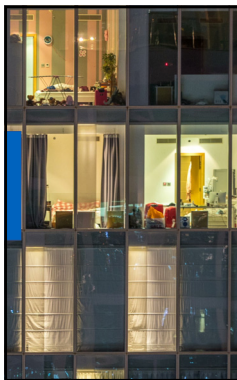
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## AIR FLOW CONTROL

- When air is ducted to multiple locations, the distribution may be tailored to the application with balancing dampers
- If the end use is variable, putting an actuated damper on one outlet will cause changes in all the other fixed damper flow rates
- Airflow control may be either **Pressure Dependent** or **Pressure Independent**
- A **Pressure Dependent** system simply opens or closes a damper in response to a control signal
- A **Pressure Independent** system has a flow sensor and control loop that responds to a control signal

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## VENTILATION DELIVERY

- Ventilation is often supplied to the low pressure side of the HVAC system; where it is then mixed and distributed
- If ventilation air is delivered directly to the space, it may conflict with other room air distribution device air streams and the interaction should be evaluated
- If the local HVAC system has a variable speed fan and the ventilation supply utilizes a distributed system, the local control damper must be pressure independent.

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## SUMMARY

- The science of air distribution is well understood in the commercial space, but in residential, it is often overlooked
- Forced air systems are best employed when the occupants are unaware of their existence – meaning no drafts or unwanted sounds
- To achieve this, all the components of a forced air system need to be understood

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## CONTACT INFORMATION

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